

Development of a Circulating Fluidized Bed for Flue Gas Carbon Capture using Solid Sorbent

James Spenik

REM Engineering Services - NETL

Overview

- A bench scale circulating fluidized bed system designated as the “Carbon Capture Unit” or “C2U” has been designed and built to evaluate the performance of sorbents on a solid substrate for removal of CO₂ from flue gases.



Program goals

- This project relates to the Existing Plants, Emissions and Capture (EPEC) Program within the Post-Combustion CO₂ Capture area and under the category of Solid Sorbents.
- Program goal: To develop fossil fuel conversion systems that can capture 90% CO₂ while keeping the increase in cost of energy service below 35%.*

** DOE/NETL Carbon Dioxide Capture and Storage R&D Roadmap, DOE/NETL Document, December 2010.*

Success criteria for sorbents: MATRIC* study

Initial performance:

30 – 50 % energy required for wet MEA process

Minimum Delta Loading:

3.0 gmol CO₂/kg sorbent

Temperature adsorption/desorption envelope:

40 – 110 C @ atm. press. with humidity

Stability, durability and performance:

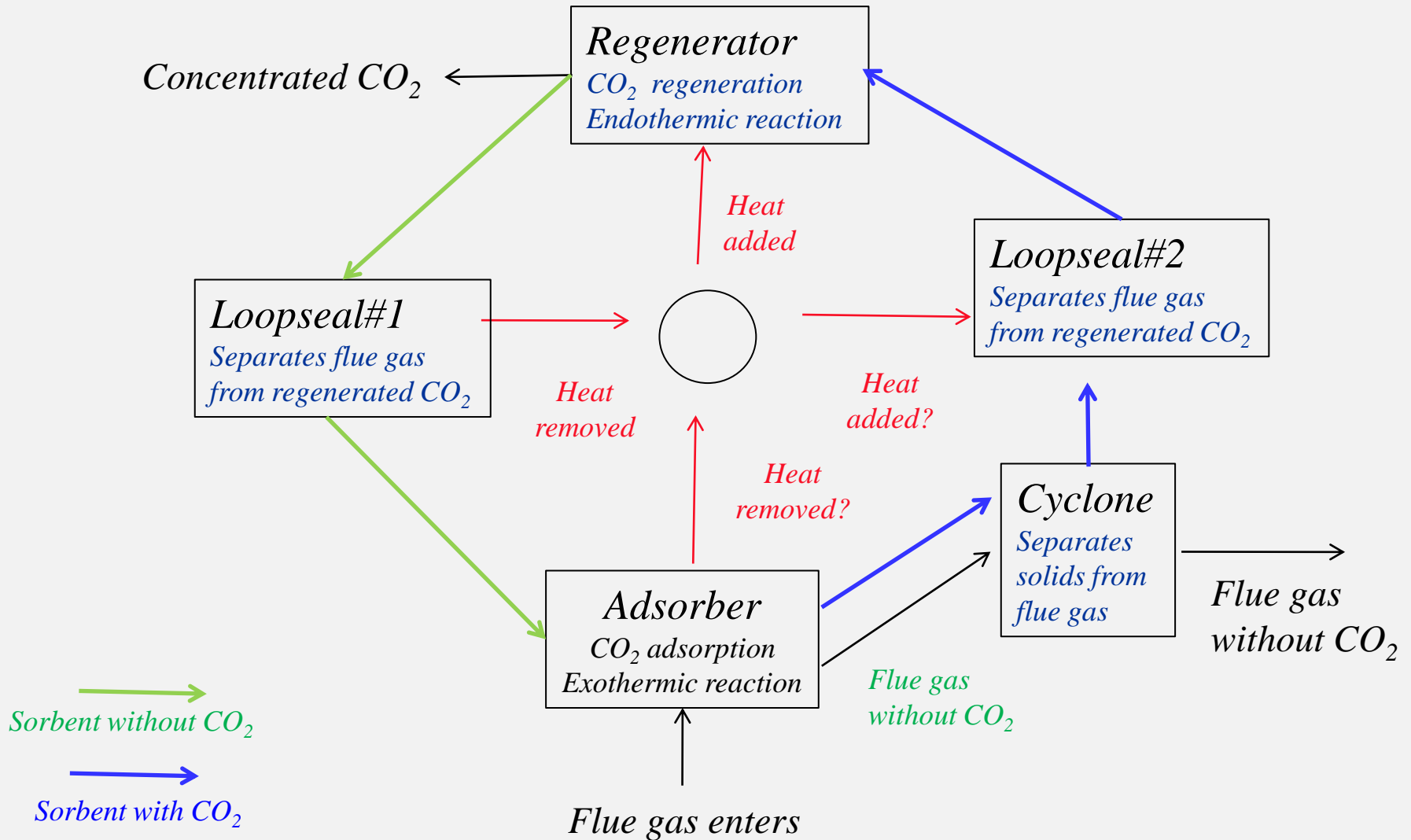
Sorbent must maintain its adsorption/desorption capability in the presence of water and other flue gases and maintain structural integrity through multiple cycles.

* Mid-Atlantic Technology, Research and Innovation Center (MATRIC), *PROCESS ANALYSES AND R&D PLANS FORWARD FOR DRY-SORBENT-BASED PROCESSES FOR REMOVAL OF CO₂ FROM POWER PLANT FLUE GASES*, July 19, 2006

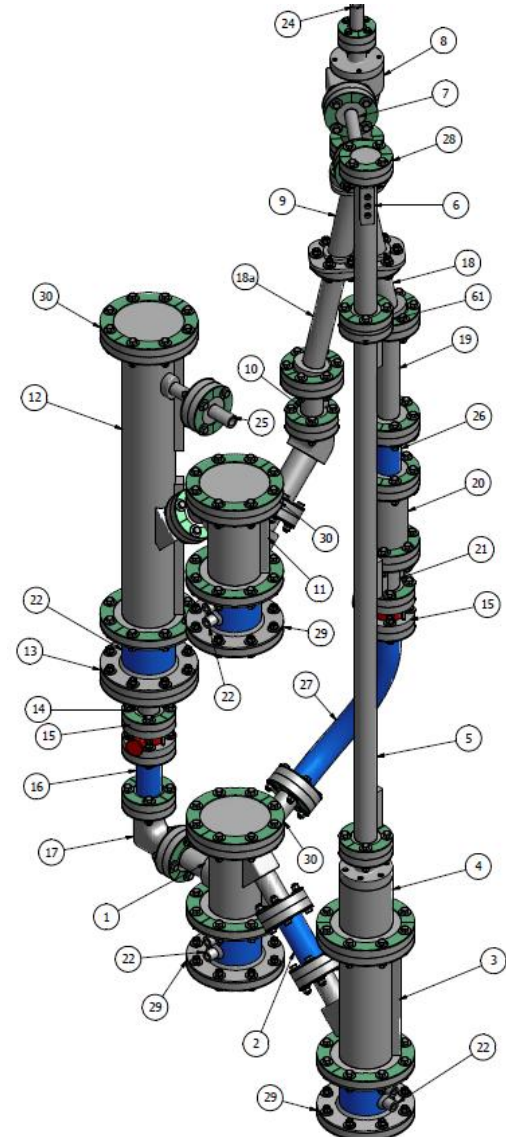
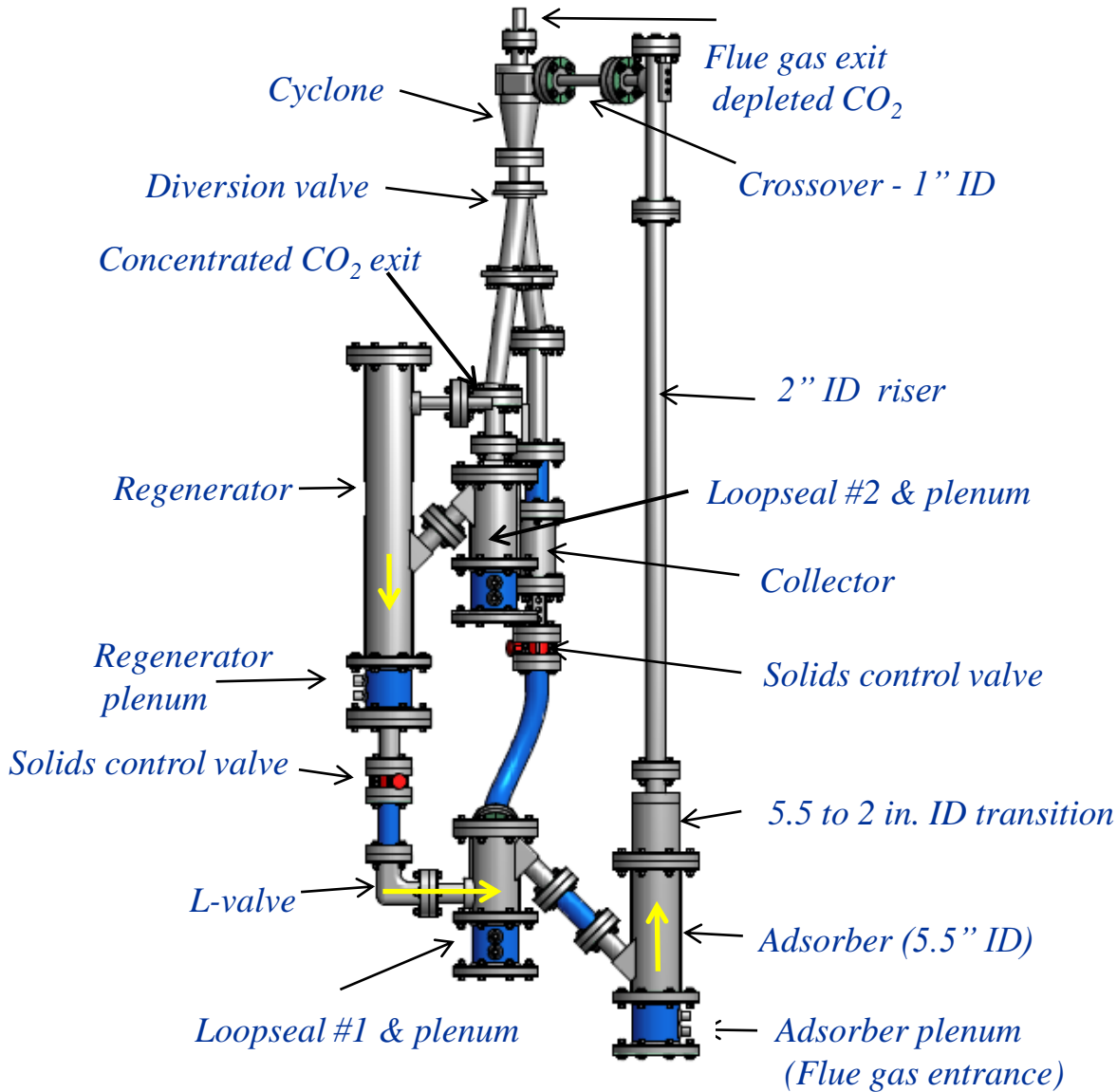
Goals/objectives of C2U

- **Flexible, inexpensive, small scale unit capable of validating CFD models including CO₂ adsorption and regeneration**
- **Desired Capabilities:**
 - Evaluate heat transfer modes
 - Define mass transfer limits
 - Map hydrodynamic parameters
 - Validate sorbent kinetics
 - Quantify desired CO₂ loading on sorbent
 - Evaluate ability to isolate processes
 - Control process variables, such as mixing, flue gas composition, residence times, etc.
 - Evaluate reactor performance

Conceptual design



C2U components



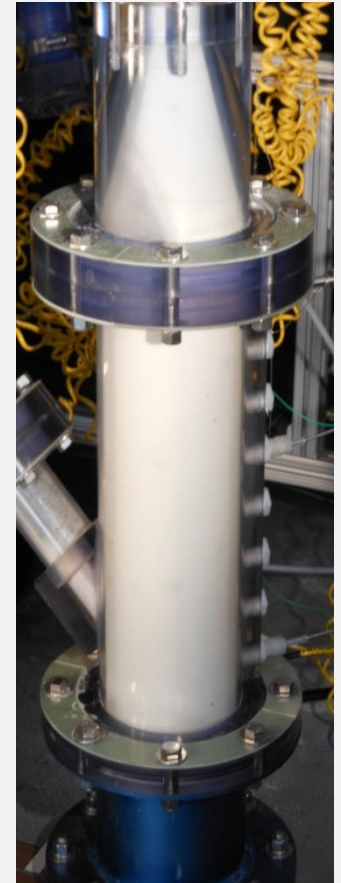
Design basis – Centerpoint of operation envelope

- **Sorbent**
 - Density: 2.0 g/cc
 - Diameter: 200 μm (Operation range : 70 to 400 μm)
 - Specific heat: 837 J/kg-K
 - Sorption capacity: 3.0 g-mol CO_2 /kg sorbent
- **Flue gas composition**
 - Dry: 81.3 % N_2 , 15.9 % CO_2 , 2.8 % O_2 , 0.0% H_2O
 - Wet: 68.1 % N_2 , 13.5 % CO_2 , 2.4 % O_2 , 15.1% H_2O
- **Flue gas flow**
 - $3.5 \times U_{\text{mf}}$ in adsorber without cooling coils (116 slpm)
 - CO_2 flow: 18.4 slpm (1.26×10^{-2} g-mol/s or 5.53×10^{-4} kg/s)
 - Equivalent to 5.0 kW (thermal) power plant. 1.75 kW with 35% efficiency

Design basis – Centerpoint of operation envelope

Adsorber – 5.5” (0.14 m) ID, 18” (0.45 m) height, polycarbonate

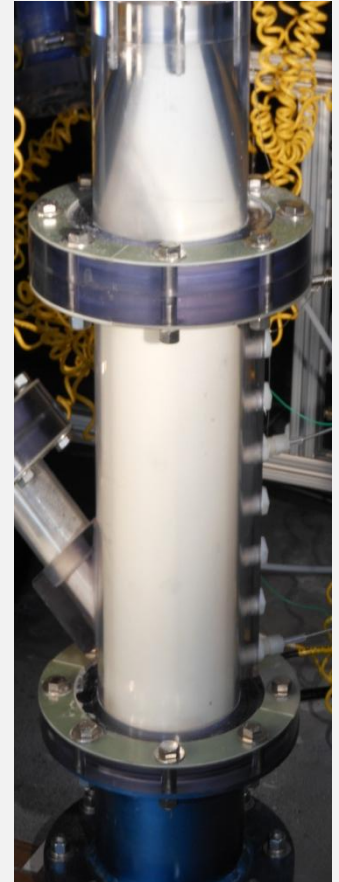
- **Sorbent circulation rate –with adsorber heat transfer**
 - CO₂ total flow: 5.53×10^{-4} kg/s (1.26×10^{-2} g-mol/s)
 - 90% CO₂ capture
 - Sorbent loading : 3.0 g-mol CO₂/kg sorbent
 - Circulation rate:
 - $0.9 * (1.26 \times 10^{-2} \text{ g-mol/s}) / (3.0 \text{ g-mol CO}_2/\text{kg sorbent}) = 3.77 \times 10^{-3} \text{ kg/s}$
(30 lb/hr)
- **Heat to be removed**
 - Heat of reaction
 - Adsorption heat of reaction ($\Delta H = 1.51 \times 10^6$ J/kg CO₂):
 - CO₂ total flow: 5.53×10^{-4} kg/s (1.26×10^{-2} g-mol/s)
 - 90% CO₂ capture
 - Heat produced: $1.51 \times 10^6 \text{ J/kg CO}_2 * 5.53 \times 10^{-4} \text{ kg/s} * 0.9 = 752 \text{ J/s}$
 - Sorbent sensible heat
 - Allow sorbent $\Delta T = 30$ °C (Enter @ 50 °C leave @ 80 °C)
 - Sorbent specific heat 837 J/kg- °C
 - $Q_{\text{sorbent}} = 0.9 * 5.53 \times 10^{-4} \text{ kg/s} * 837 \text{ J/kg- }^\circ\text{C} * 30 \text{ }^\circ\text{C} = 95 \text{ J/s}$
 - Heat to be removed = $752 - 95 = 657 \text{ J/s}$



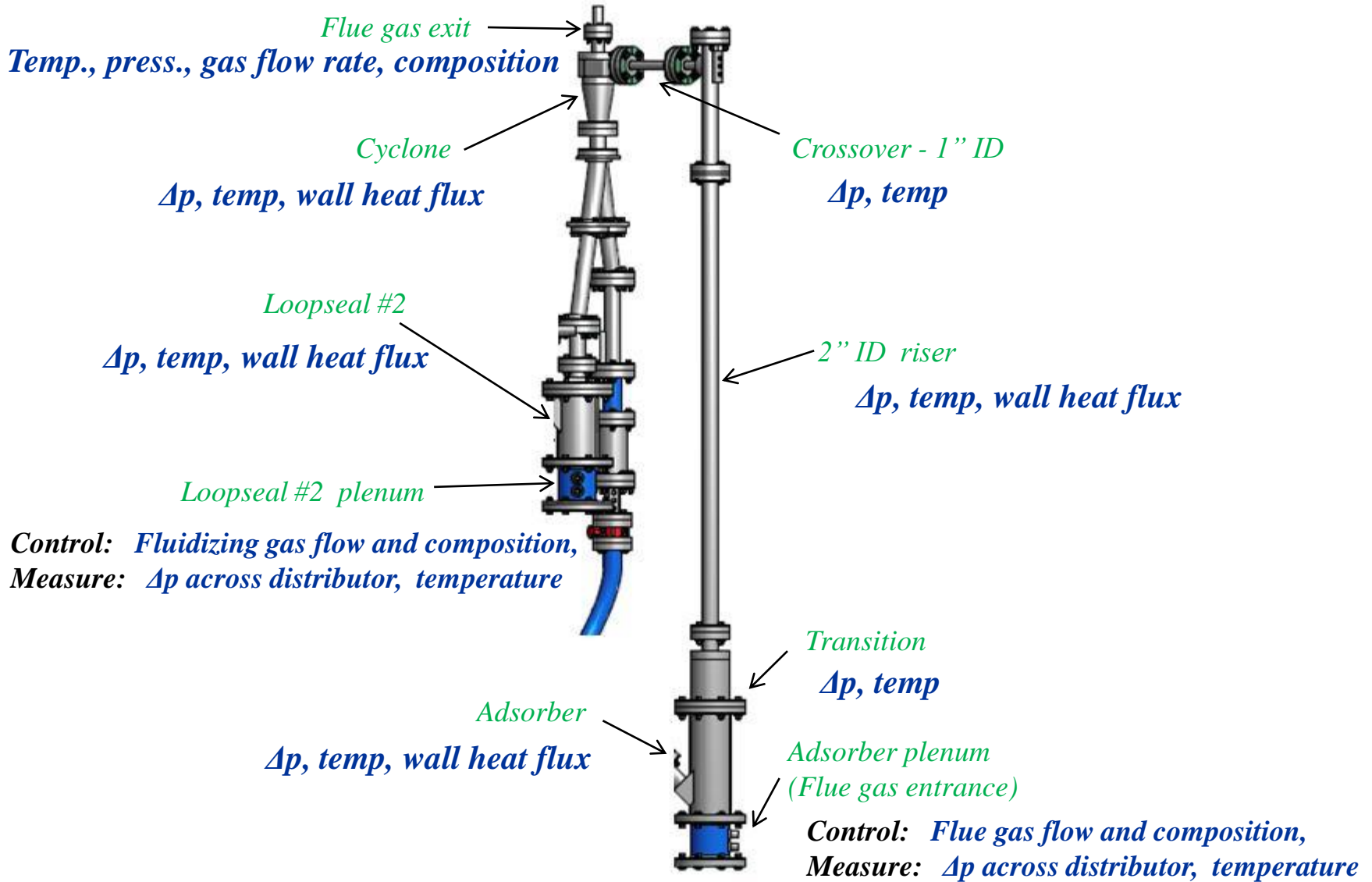
Design basis – Centerpoint of operation envelope

Adsorber – 5.5'' (0.14 m) ID, 18'' (0.45 m) height, polycarbonate

- **Required sorbent circulation rate - without adsorber heat transfer**
 - Heat produced by reaction 752 J/s
 - Sorbent circulation rate
 - Sorbent specific heat: 837 J/kg- C
 - Assume sorbent temp not permitted to exceed 80 C for adsorption, enters adsorber at 50 C ($\Delta T = 30$ C)
 - Circulation rate: $(752 \text{ J/s}) / (30 \text{ C} * 837 \text{ J/kg-K}) = 3.0 \times 10^{-2} \text{ kg/s}$ (237 lb/hr)
 - Sorbent loading
 - $0.9 * (1.26 \times 10^{-2} \text{ g-mol/s}) / 3.0 \times 10^{-2} \text{ kg/s} = 0.4 \text{ g-mol CO}_2/\text{kg sorbent}$
 - Some heat is removed by flue gas flow (approx. 10%) decreasing the required sorbent circulation rate calculated above



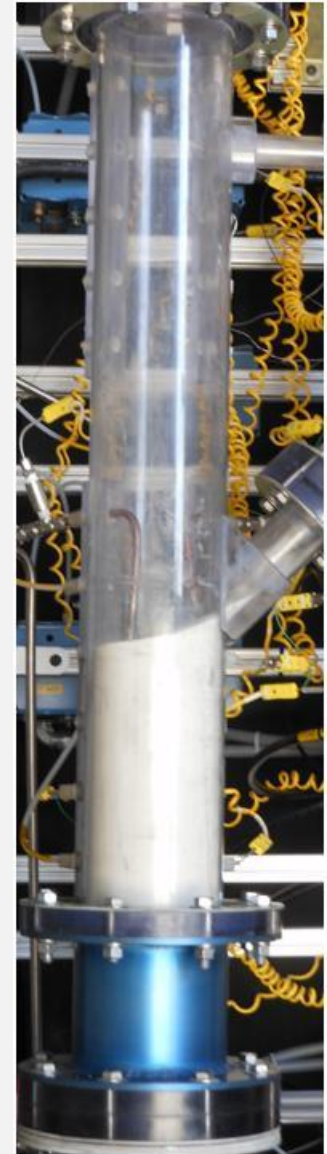
Measurements-Adsorber side



Regeneration

Regenerator – 5.5” (0.14 m) ID, 40” (0.45 m) height, polycarbonate with internal coils

- **Heat addition for 100 % CO₂ regeneration**
 - Reverse amine reaction - 752 J/s
 - Heat substrate from 80 to 100 °C
 - Circulation rate: 3.77×10^{-3} kg/s $Q_{\text{sorbent}} = 63$ J/s Total = 815 J/s
 - Circulation rate: 3.00×10^{-2} kg/s $Q_{\text{sorbent}} = 501$ J/s Total = 1253 J/s



Heat transfer coils

Heat transfer within all reactors accomplished using two copper coils nested axisymmetrically.

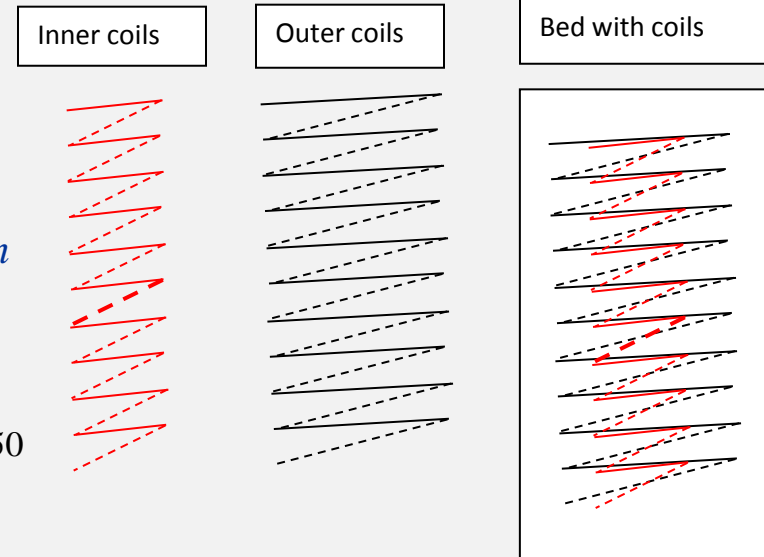
Chilled water flows through the coils for cooling. Heat addition utilizes heated oil.

Heat transfer coefficient is determined using correlation from *Vreedenburg 1958 for horizontal tubes in a bubbling bed

$$\frac{h_c D_t}{k_g} = 0.66 \text{Pr}_g^{0.3} \left(\frac{\rho_s \epsilon}{\rho_g \epsilon} \right)^{0.44} \text{Re}_D^{0.44} \quad \text{when} \quad \frac{\rho_s}{\rho_g} \text{Re}_p \leq 2050$$

where

$$\text{Re}_p = \frac{d_p \rho_g U_g}{\mu_g} \quad \text{Re}_D = \frac{D_t \rho_g U_g}{\mu_g}$$



*Handbook of Fluidization and Fluid Particle Systems (Yang) p. 263

Heat addition by coils in regenerator

Using values at right and Vreedenburg correlation the following heat transfer coefficients are determined:

Heat transfer coeff of outer tube (h) = 188.6 W/m²-K
 $hA_{outer} = 28.8\text{W/K}$

Heat transfer coeff of inner tube (h) = 211.8W/m²-K
 $hA_{inner} = 17.52\text{W/K}$

$U_g = 2*U_{mf} = 0.064 \text{ m/s}$
 $\rho_{gas} = 1.28 \text{ kg/m}^3 \text{ @ operating pressure}$
 $\mu_{gas} = 1.81 \times 10^{-5} \text{ kg/m-s}$
 $k_{gas} = 2.58 \times 10^{-2} \text{ W/m-K}$
 $Pr = 0.713$
 Void fraction = 0.516
 Outer coil tube diameter = $1.27 \times 10^{-2} \text{ m}$
 Outer coil tube length = 3.83 m
 Outer coil tube surface area = 0.229 m²
 Inner coil tube diameter = $9.67 \times 10^{-3} \text{ m}$
 Inner coil tube length = 2.55 m
 Inner coil tube surface area = 0.116 m²

Required Log Mean Temperature Difference

$$\Delta T = \frac{Q}{hA_{outer} + hA_{inner}}$$

For circulation rate $3.77 \times 10^{-3} \text{ kg/s}$ $\Delta T = 17.6 \text{ K}$ since $Q = 815 \text{ J/s}$

For circulation rate $3.00 \times 10^{-2} \text{ kg/s}$ $\Delta T = 27.0\text{K}$ since $Q = 1253 \text{ J/s}$



Heat addition by coils in regenerator

Log mean temperature difference for a parallel flow heat exchanger

$$LMTD = \frac{\Delta T_{x=0} - \Delta T_{x=L}}{\ln \left(\frac{\Delta T_{x=0}}{\Delta T_{x=L}} \right)} = \frac{(T_{h,i} - T_{c,i}) - (T_{h,o} - T_{c,o})}{\ln \left(\frac{T_{h,i} - T_{c,i}}{T_{h,o} - T_{c,o}} \right)}$$

$T_{h,i}$ - Temperature of oil at the inlet.
 $T_{c,i}$ - Temperature of sorbent at the inlet.
 $T_{h,o}$ - Temperature of oil at the outlet.
 $T_{c,o}$ - Temperature of sorbent at the outlet.

Increase sorbent temperature from 80 C ($T_{c,i}$) to 100 C ($T_{c,o}$) and add regeneration energy

Using heated oil @ 4.0 liter/min (7.33×10^{-2} kg/s) sp. ht. = 1910 J/kg-K

$$\Delta T_{oil} = Q / (\text{mass flow} * \text{sp. ht.})$$

Low circulation rate:

$$\Delta T_{oil} = -5.8 \text{ C for LMTD} - 17.6 \text{ C}$$

Oil inlet temp = 113.6 C, Oil exit temp = 107.8 C - Using LMTD equation:

High circulation rate:

$$\Delta T_{oil} = -9.0 \text{ C for LMTD} - 27.0 \text{ C} - \text{sorbent circ rate } 3.00 \times 10^{-2} \text{ kg/s}$$

Oil inlet temp = 124.0 C, Oil exit temp = 115.0 C

Heat removal by coils in Loopseal #1

Decrease sorbent temperature from 80 C ($T_{h,i}$) to 50 C ($T_{h,o}$)

Heat transfer coeff of outer tube (h) = 214 W/m²-K, hA_{outer} = 24.6 W/K,

Heat transfer coeff of inner tube (h) = 252 W/m²-K, hA_{inner} = 14.5 W/K

$Q_{low\ circ}$ = 158 J/s, LMTD = 4.0 C, $Q_{high\ circ}$ = 1253 J/s, LMTD = 32.1 C

Using chilled water @ 4.0 liter/min (6.67×10^{-2} kg/s)

sp. ht. = 4181 J/kg-K

*$\Delta T_{water} = Q / (\text{mass flow} * \text{sp. ht.})$*

Low circulation rate:

$\Delta T_{water} = 0.6$ C for LMTD - 4.0 C

Water inlet temp = 49.4 C,

water exit temp = 50.0 C

High circulation rate:

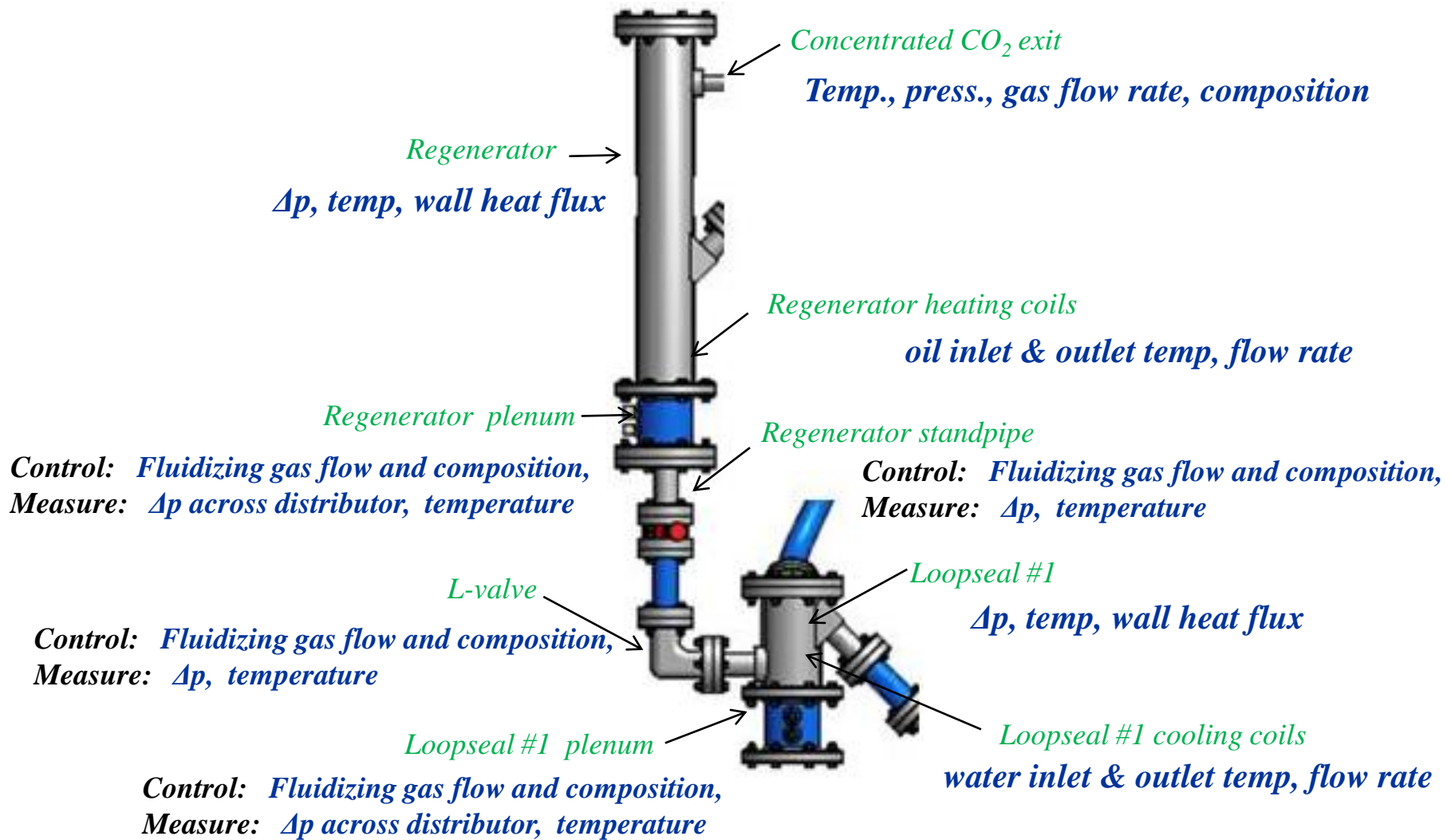
$\Delta T_{water} = 4.5$ C for LMTD - 32.1 C

Water inlet temp = 33.2 C,

water exit temp = 37.7 C



Measurements- Regenerator side



Conclusions

The design meets the criteria previously expressed

- Flexible, inexpensive, small scale unit capable of validating CFD models including CO₂ adsorption and regeneration
- **Desired Capabilities:**
 - Evaluate heat transfer modes
 - Define mass transfer limits
 - Map hydrodynamic parameters
 - Validate sorbent kinetics
 - Quantify desired CO₂ loading on sorbent
 - Evaluate ability to isolate processes
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