

**A Two Grid Formulation for Fluid-Particle
Systems using the Discrete Element Method.**

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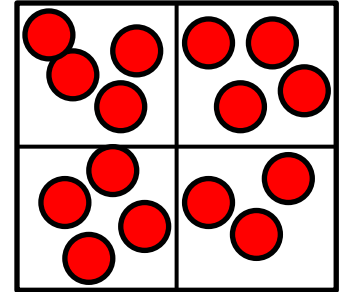


Outline

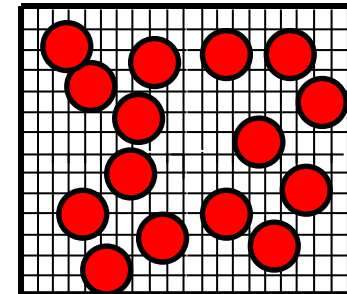
- **Motivation**
- **Objectives**
- **Methodology**
- **Two Grid Formulation**
- **Results**
- **Conclusion**

Motivation

- **Conventional DEM-CFD framework uses a single grid approach.**
- **Particles require coarser grid to maintain smoothness in local solid volume fraction and avoid instability.**
- **Fine scale fluid features like turbulence, wall shear stress and heat transfer coefficient at immersed surfaces require finer grids for better resolution.**
- **Difficult to resolve flows with large sized particles and high flow velocities like jets in jetting fluidized beds.**
- **Difficult to resolve small geometrical features influencing the flow with conventional DEM-CFD framework.**



Coarse Grid



Fine Grid

Objectives

- **Develop and implement the two-grid framework for DEM-CFD in our in-house code GenIDLEST.**
- **Perform and validate two-grid simulations, not possible with single grid framework on jetting fluidized bed setup.**
- **Avoid instability faced in single grid framework for flow conditions like jets, if the particle size becomes comparable to jet size in jetting fluidized beds.**

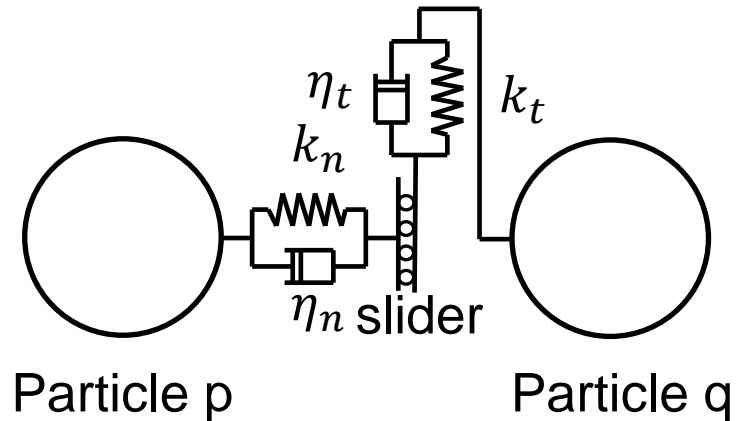
Methodology

- **Soft Sphere DEM**

- **Linear Spring-Mass Dashpot**

Normal Force $\vec{f}_{n,pq} = -k_n \vec{\delta}_{n,pq} - \eta_n \vec{v}_{n,pq}$

Tangential Force $\vec{f}_{t,pq} = -k_t \vec{\delta}_{t,pq} - \eta_t \vec{v}_{t,pq}$



- **Model B (Gidaspow)**

Continuity $\nabla \cdot (\epsilon \rho_g \vec{u}) = 0$

Momentum $\frac{\partial(\epsilon \rho_g \vec{u})}{\partial t} + \nabla \cdot (\epsilon \rho_g \vec{u} \vec{u}) = -\nabla p + \nabla \cdot (\epsilon \vec{\tau}_g) - \frac{1}{V_{fluid,cell}} \sum_N \frac{V_p \beta}{(1-\epsilon)} (\vec{u} - \vec{v}_p) + \rho_g \vec{g}$

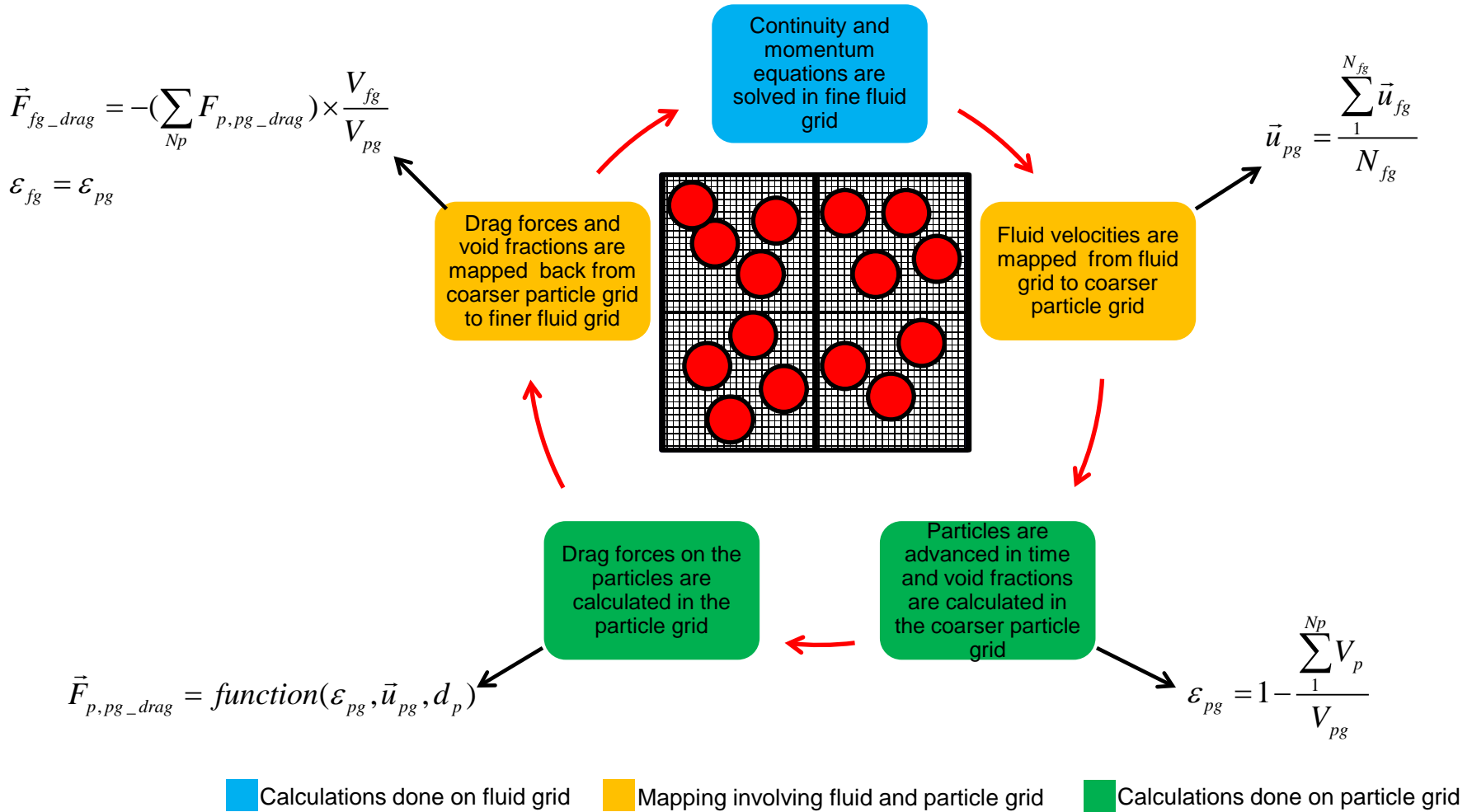
- **Ergun, Wen & Yu Drag Correlations**

Ergun, 1952 $\beta = 150 \frac{(1-\epsilon)^2 \mu_g}{\epsilon d_p^2} + 1.75(1-\epsilon) \frac{\rho_g}{d_p} |\vec{u} - \vec{v}_p|$ if $(\epsilon < 0.8)$

Wen and Yu, 1966 $\beta = \frac{3}{4} C_D \frac{\epsilon(1-\epsilon)}{d_p} \rho_g |\vec{u} - \vec{v}_p| \epsilon^{-2.65}$ if $(\epsilon \geq 0.8)$

- **Fractional Step Time Marching**

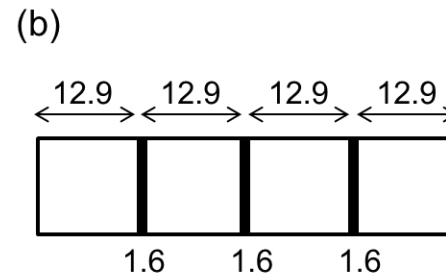
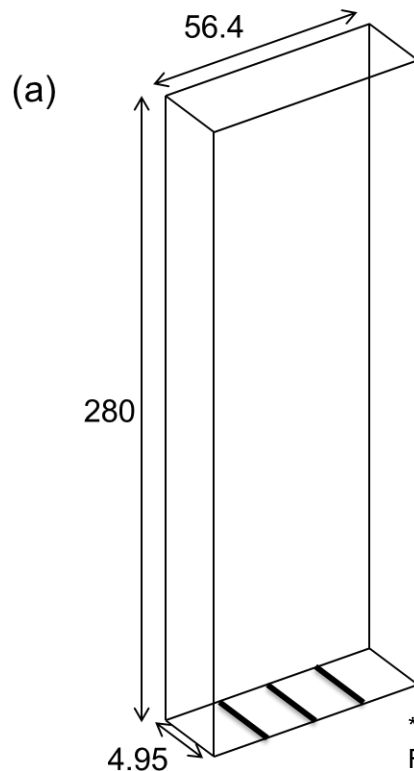
Two-grid formulation



Flow diagram of fluid-particle grid mapping in two-grid framework

Computational Details

- 3D simulations performed on a small lab scale fluidized bed setup*.



Distributor plate design (Dimensions are in mm)

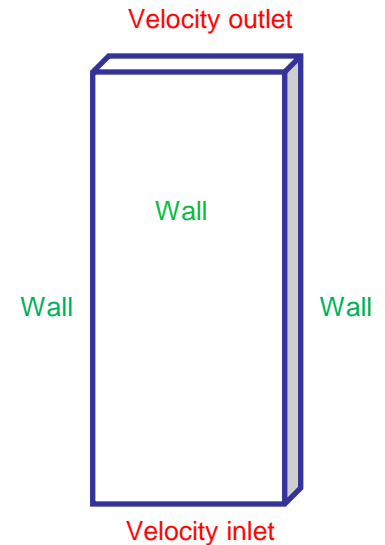
*Brown, L. S. and Y. B. Lattimer, "Experimental Hydrodynamics of Multiple Jet Systems in a Fluidized and Spouted Bed." International Journal of Multiphase Flow, submitted.

Particle and DEM Properties

Particle properties	
Material	Glass
Diameter	750 microns
Number of particles	50,000
Density	2500 kg/m ³
Coefficient of friction	0.10
Minimum fluidization velocity	0.43 m/s
Spring constant (k_n, k_t)	800 N/m
Coefficient of restitution	0.90

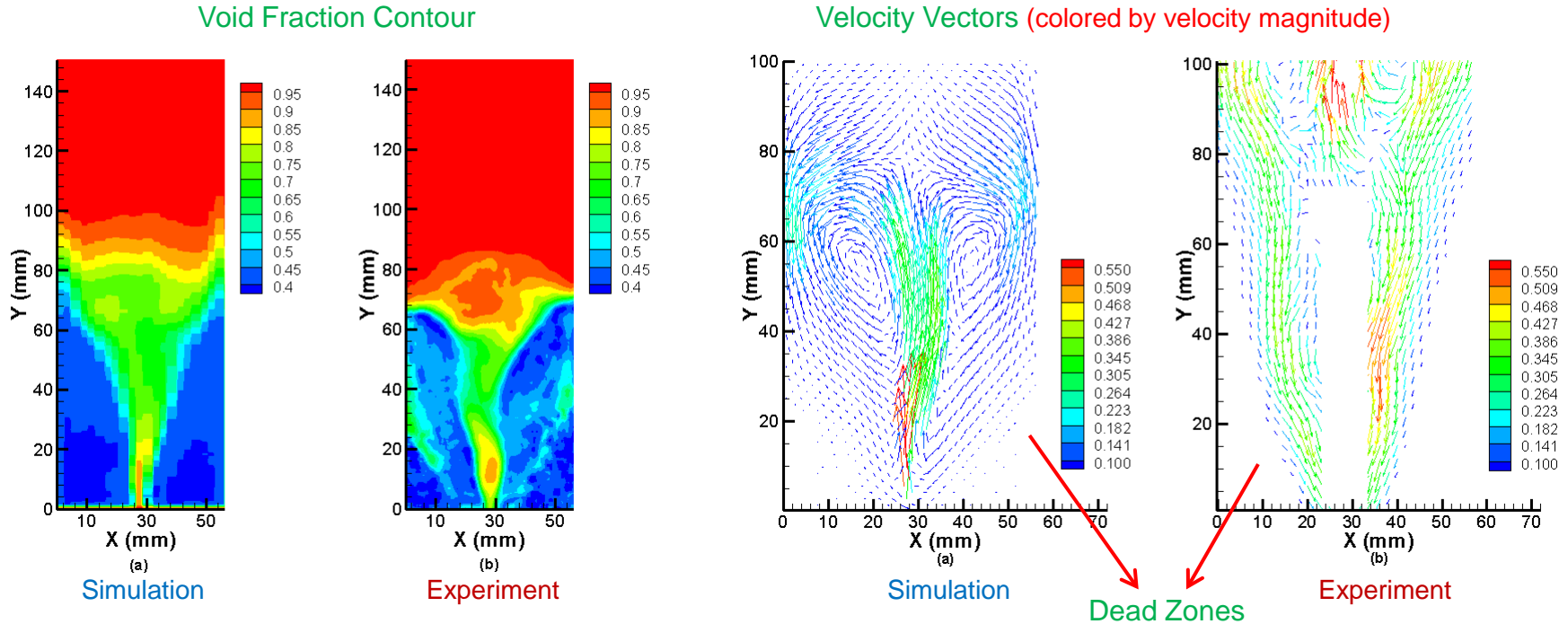
Grid Details

Jet details	
Number of jets	1,2 and 3
Superficial velocity	0.699 m/s($1.60U_{mf}$), 1.294 m/s($3U_{mf}$)
Jet width	1.6 mm
Fluid grid details	
Along height	200 cells ($\Delta y=1.40$ mm)
Along width	175 cells ($\Delta x=0.322$ mm)
Along depth	4 cells ($\Delta z=1.2375$ mm)
Particle grid details	
$\Delta x = \Delta y = \Delta z = 3d_p = 2.25$ mm	Fluid grid size for single grid framework



Results – Single Jet 1.60 U_{mf}

(Time averaged for 3 s)

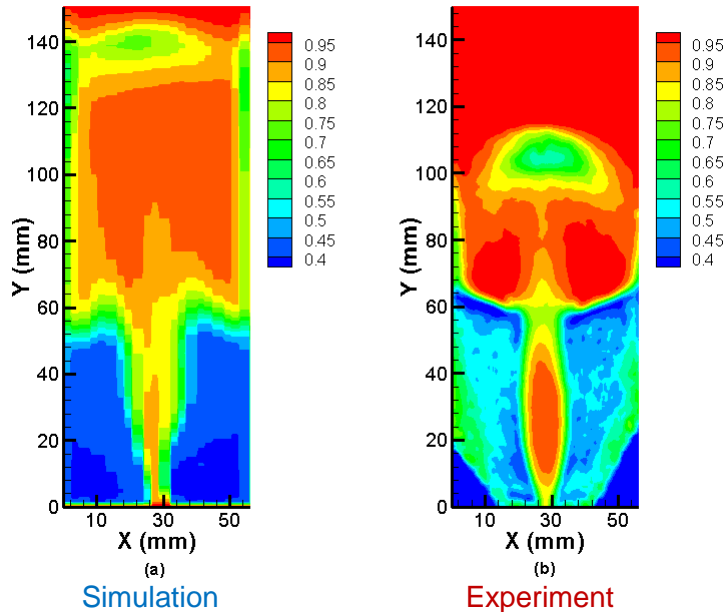


- The results are time averaged for 3 seconds after the first 5 seconds.
- The experimental velocity vectors inside the jet are masked as the PIV resolution is not sufficient to capture the high particle velocities inside the jet.
- Void fractions compare closely to the experiment.
- Dead zones are higher in experiment compared to simulation.

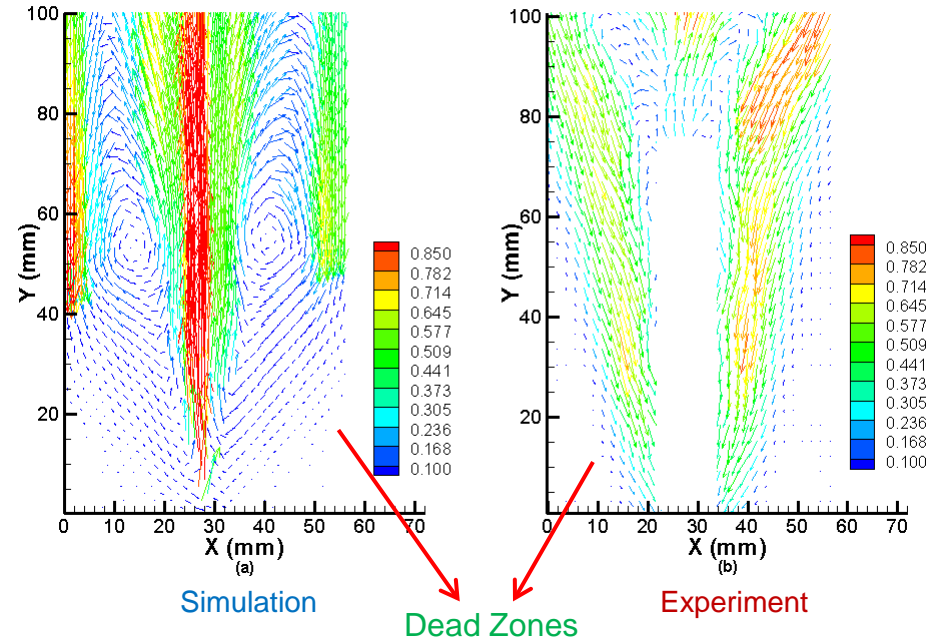
Results – Single Jet 3.0 U_{mf}

(Time averaged for 3 s)

Void Fraction Contour



Velocity Vectors (colored by velocity magnitude)



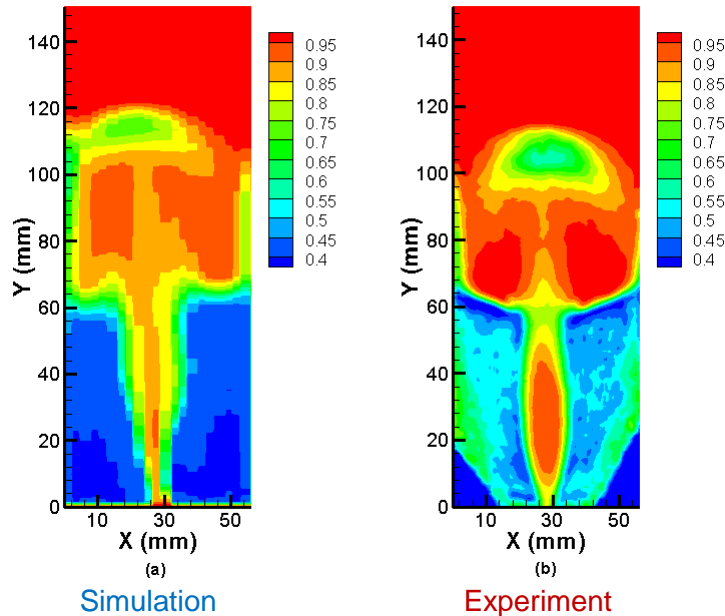
- Fountain formation can be seen in both the experiment and simulation.
- The simulation over predicts the fountain height.
- This is due to higher particle-wall friction present in experiment.
- High central particle velocities can be seen in the simulation.

Results – Single Jet

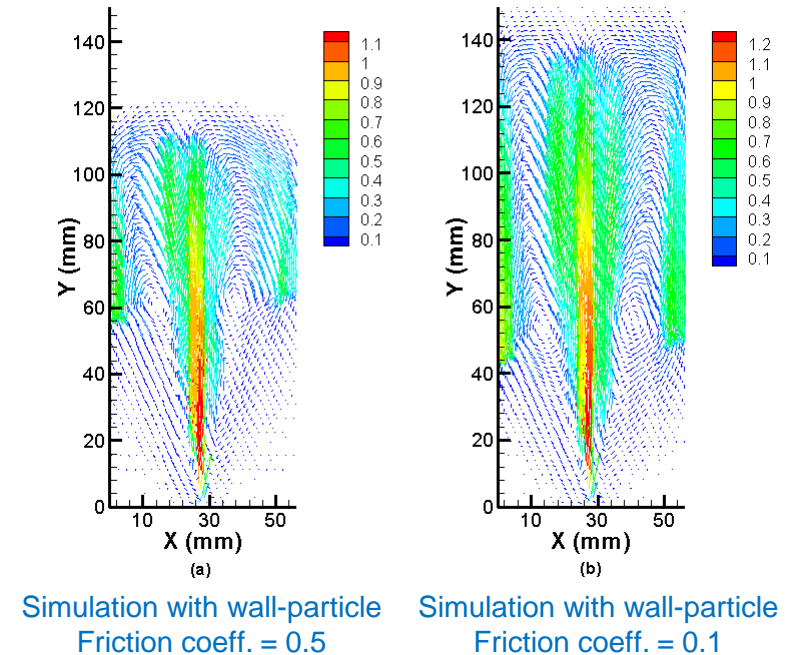
3.0 U_{mf} with higher wall friction

(Time averaged for 3 s)

Void Fraction Contour



Comparison of Velocity Vectors (colored by velocity magnitude)

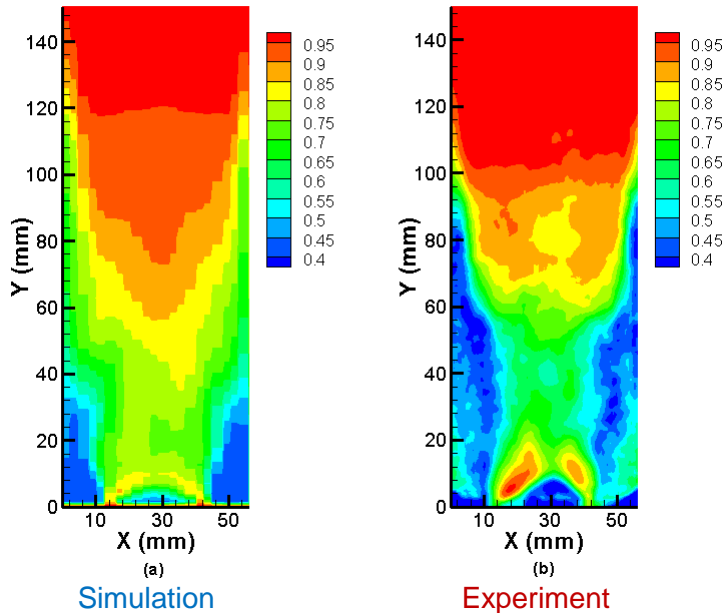


- A single case to test the effect of wall-particle friction has been shown.
- The wall-particle friction coefficient has been increased to 0.5 from 0.1.
- The cluster of particles/fountain has come down and compares closely with experiment.
- A slightly higher dead zone formation can be observed in the velocity vectors.

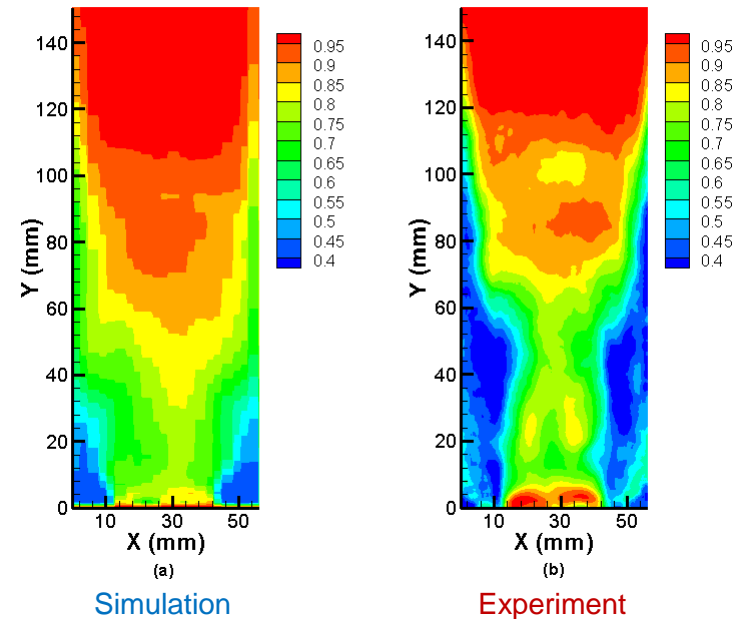
Results – Multiple Jets 3.0 U_{mf}

(Time averaged for 3 s)

Void Fraction Contour (Double Jets)



Void Fraction Contour (Triple Jets)



- Experimental PIV is not suited to capture velocity vectors for multiple jets.
- Void fraction profiles compare well with the experiments.
- Parabolic void fraction profiles can be seen for both the simulations and experiments.
- Lower wall-particle friction leads to smaller dead zones in the simulations.
- Bed expansion is comparable with the experiments.

Conclusions

- The commonly used single grid approach in coupled CFD-DEM calculations is limited by the requirement of having coarser fluid grid for stability.
- To overcome this limitation, a two-grid method has been implemented and tested on jetting fluidized beds.
- For the 3 U_{mf} and single jet case, DEM predicts a higher fountain height compared to the experiment.
- Much better agreement of the predicted fountain height with the experiment is achieved with a higher wall-particle friction coefficient.
- Overall, the trends predicted by the two-grid scheme are in agreement with experimental observations, particularly for multiple jets. The single grid framework was tried with this setup and it became unstable.

THANK YOU
Questions?