

Ongoing Validation Efforts in Gas-Liquid Flows with Phase Change in System and CFD Codes

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Overview

- Two-fluid model
- Challenges
- Experimental facility
- Multiphase instrumentation
- Validation approach
- Other ongoing experiments



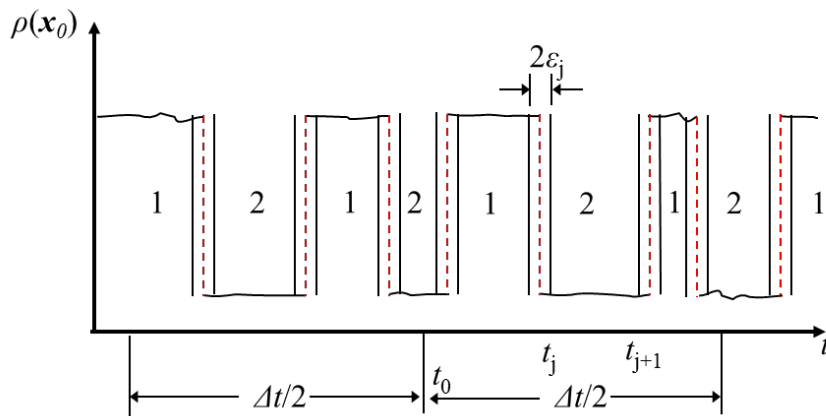
Two-fluid model with IATE

Two-fluid Model:

$$\frac{\partial(\alpha_k \rho_k)}{\partial t} + \nabla \cdot (\alpha_k \rho_k \mathbf{v}_k) = \Gamma_k$$

$$\frac{\partial(\alpha_k \rho_k \mathbf{v}_k)}{\partial t} + \nabla \cdot (\alpha_k \rho_k \mathbf{v}_k \mathbf{v}_k) = -\alpha_k \nabla p_k + \nabla \cdot [\alpha_k (\overline{\boldsymbol{\tau}}_k + \boldsymbol{\tau}_k^T)] + \alpha_k \rho_k \mathbf{g}_k + \mathbf{v}_{ki} \Gamma_k + \mathbf{M}_{ik}$$

$$\frac{\partial(\alpha_k \rho_k h_k)}{\partial t} + \nabla \cdot (\alpha_k \rho_k \mathbf{v}_k h_k) = -p_k \frac{D_k}{Dt} \alpha_k + \nabla \cdot [\alpha_k (\overline{q}_k + q_k^T)] + a_i q_{ki}'' + \phi_k + h_{ki} \Gamma_k$$



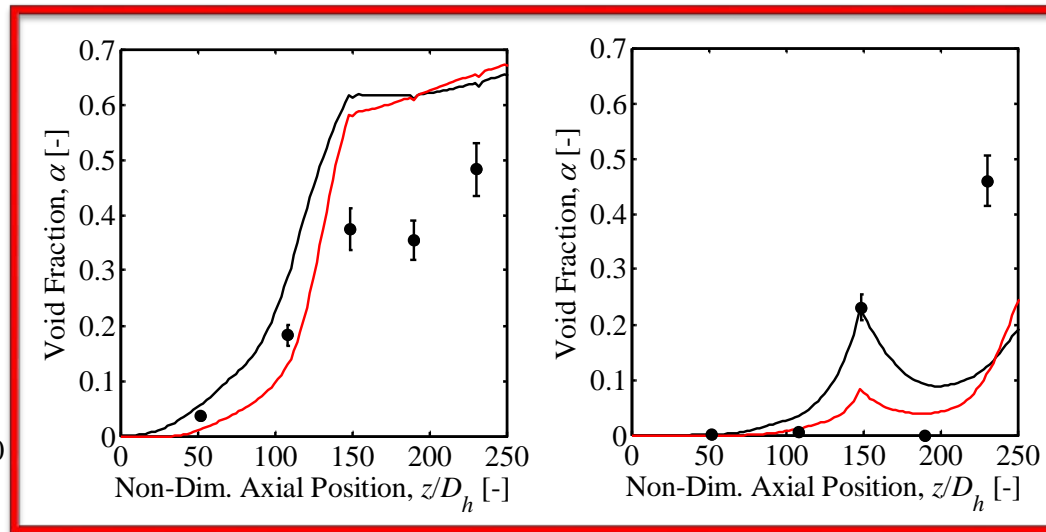
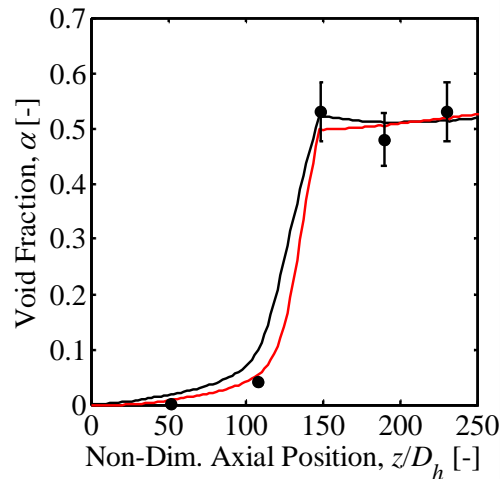
$I_k = a_i \times \text{Mean Interfacial Driving Flux}$

$$a_i \equiv \frac{1}{\Delta t} \sum_j \frac{1}{v_{ni,j}}$$

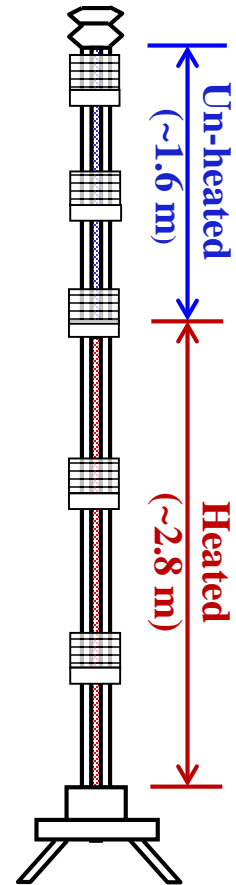
Challenges in 1-D codes

- Nuclear system codes ‘built’ for high pressure, high flow

Fullmer et al., *PNE*, 2016



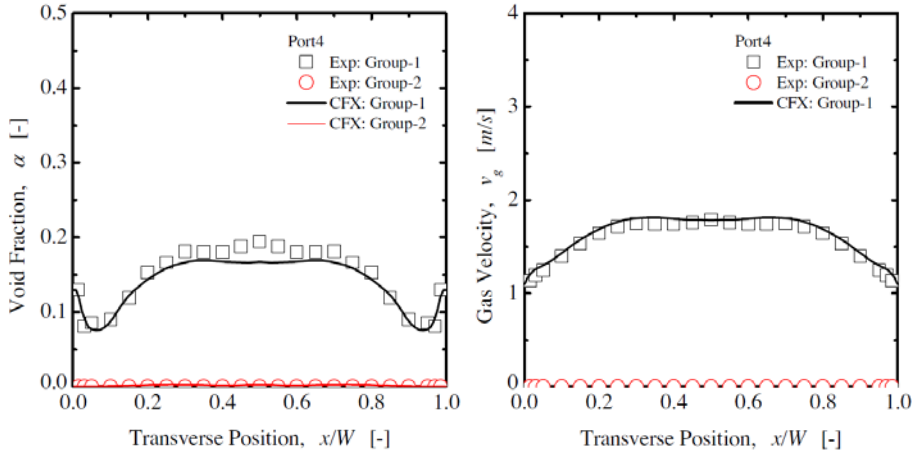
- Large error in void fraction under low pressure, low flow
- Passive safety systems
- Natural circulation



Challenges in multi-D codes

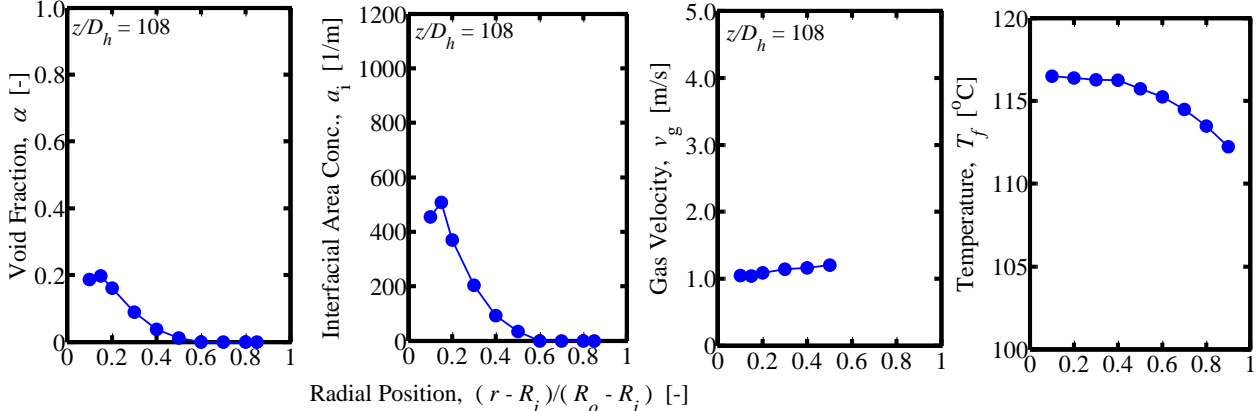
- Phase distribution – Interfacial forces

Sharma et al., *NED*, 2017

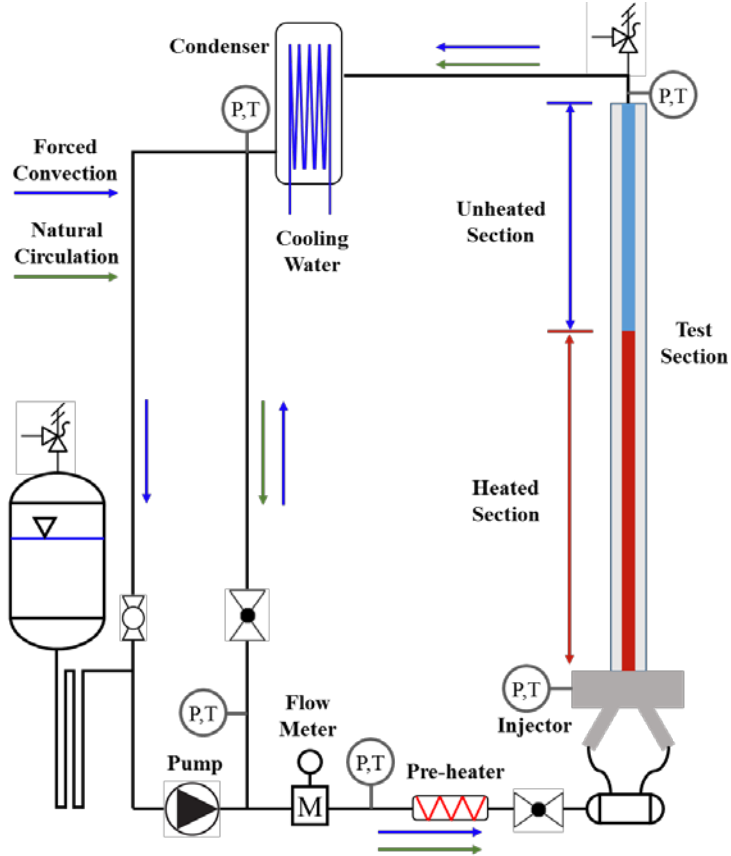


- Phase-change

- Wall nucleation
- Condensation
- Low quality CHF



Experimental Capability: Heated annulus

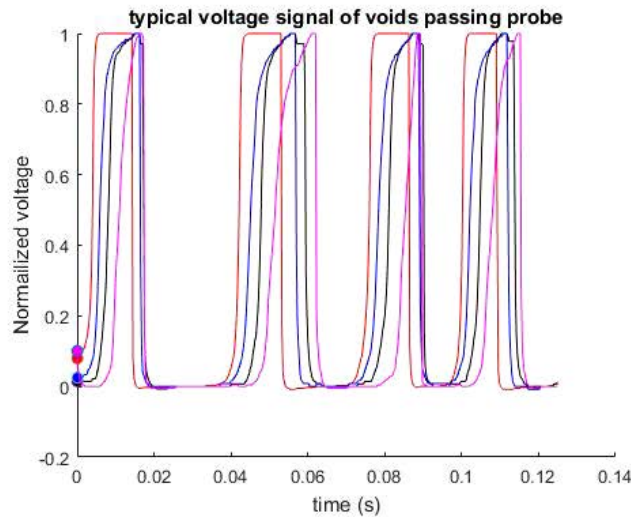
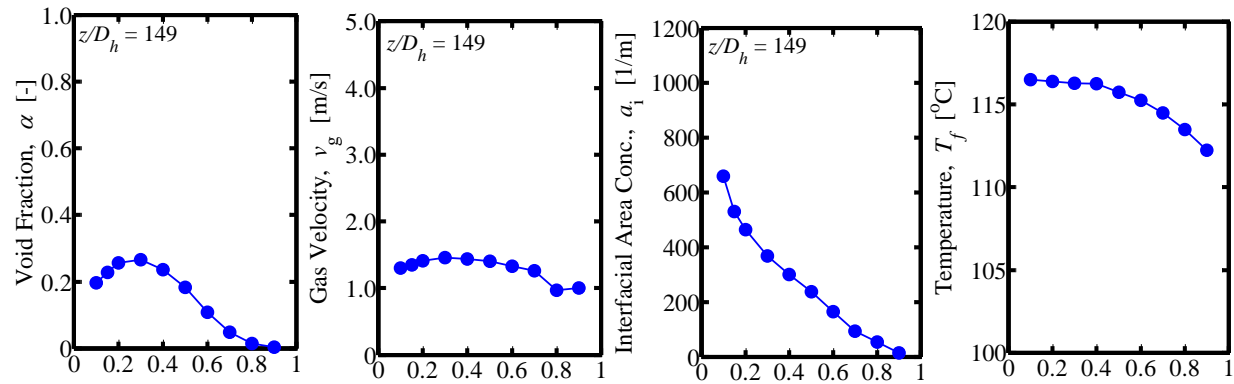


Facility Specifications

Geometry	Vertical internally-heated annulus
Channel Length	5.03 m
Heated length	3 m
Annular gap	3.81 cm (D_o), 1.91 cm (D_i)
Pressure (gauge)	0 ~ 1 MPa
Heat Flux	0 – 300 kW/m ²
Velocity	0 ~ 5 m/s



Experimental Capability: Heated annulus

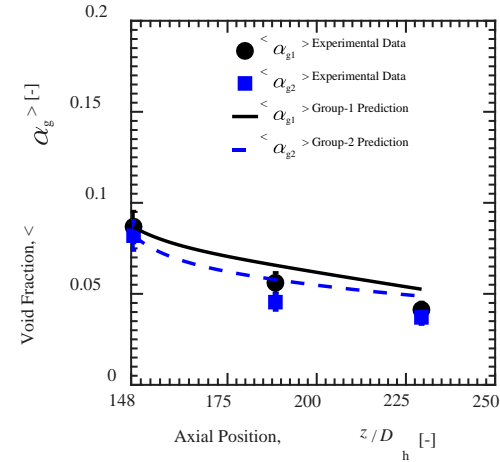
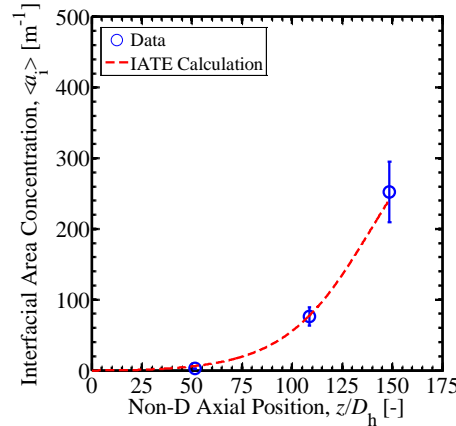


$$a_i \equiv \frac{1}{\Delta t} \sum_j \frac{1}{v_{ni,j}} = \sum_j a_{ij}$$

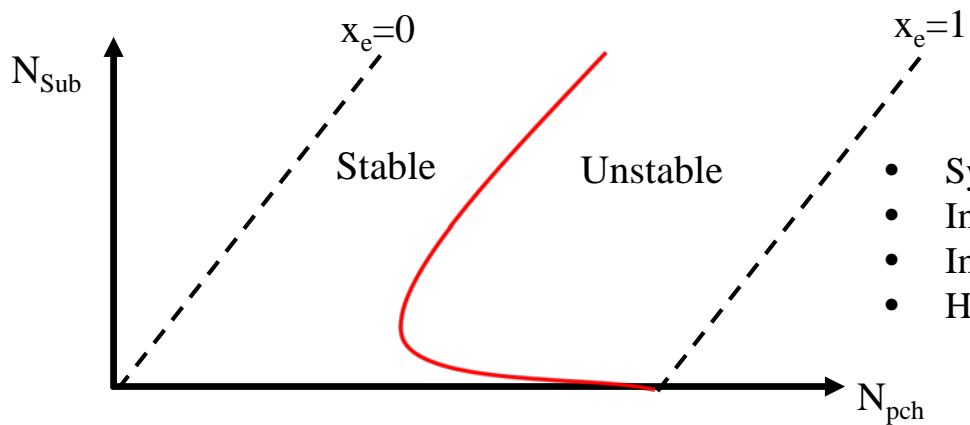
Experimental domain

➤ Model development and benchmark

- Subcooled boiling
- Condensation



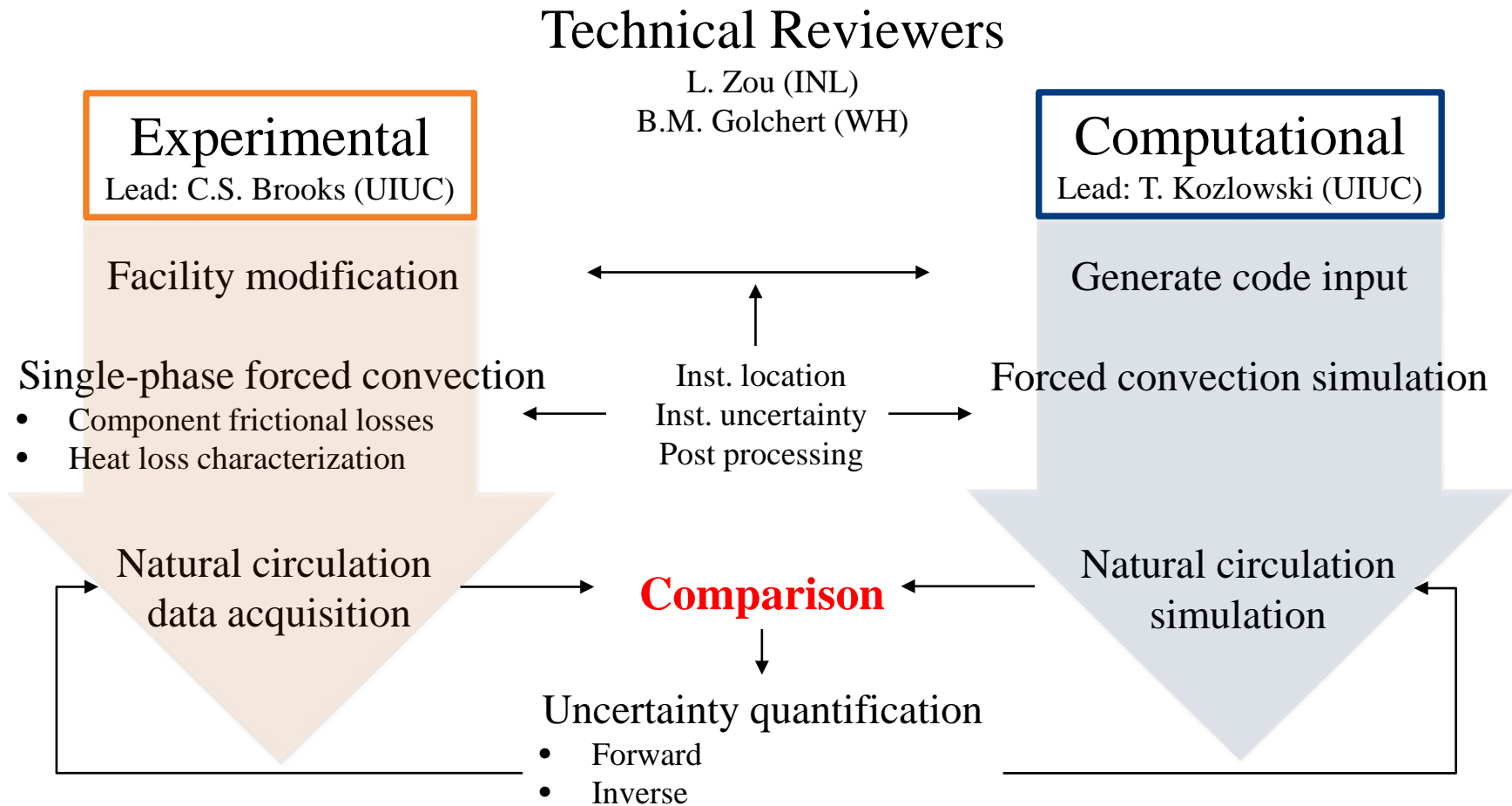
➤ System code validation: DOE- NEUP: Natural circulation



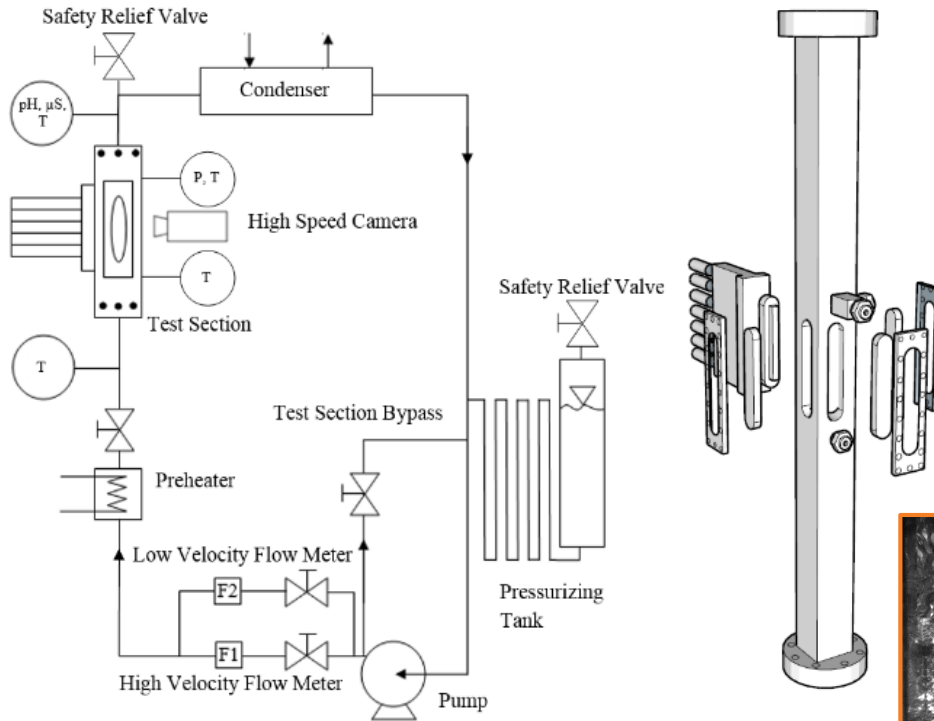
- System pressure
- Inlet temperature
- Inlet k-loss
- Heat flux



Validation approach- NEUP



Multiphase Thermo-fluid Dynamics Lab



- Critical heat flux measurement
- Wall nucleation measurement
- Effect of gap thickness on boiling heat transfer
- Sub-atmospheric boiling
- High speed imaging, pressure, temperature.
- Boiling surface engineering (coatings, laser-texturing, wettability patterning, etc.)

<i>Facility Specifications</i>	
Geometry	<i>Vertical rectangular channel</i>
Channel Dimensions	<i>99.8 cm x 1.27 cm x 1.27 cm (variable)</i>
Heated Surface	<i>10.8 cm x 1.27 cm</i>
Pressure (gauge)	<i>-0.08 ~ 1 MPa</i>
Heat Flux	<i>0 ~ 2 MW/m²</i>

